# Local Element Operations for Curved Simplex Meshes

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## SUMMARY

Mesh optimization procedures are generally a combination of node smoothing and discrete operations which affect a small number of elements to improve the quality of the overall mesh. These procedures are useful as a post-processing step in mesh generation procedures and in applications such as fluid simulations with severely deforming domains. In order to perform high-order mesh optimization, these ingredients must also be extended to high-order (curved) meshes. In this work, we present a method to perform local element operations on curved meshes. The mesh operations discussed in this work are edge/face swaps, edge collapses, and edge splitting (more generally refinement) for triangular and tetrahedral meshes. These local operations are performed by first identifying the patch of elements which contain the edge/face being acted on, performing the operation as a "straight-sided one" by placing the high-order nodes via an isoparametric mapping from the master element, and smoothing the high-order nodes on the elements in the patch by minimizing a Jacobian-based high-order mesh distortion measure. Since the initial "straight-sided guess" from the placement of the nodes via the isoparametric mapping frequently results in invalid elements, the distortion measure must be regularized which allows for mesh untangling for the optimization to succeed. We present several examples in 2D and 3D to demonstrate these local operations and how they can be combined with a high-order node smoothing procedure to maintain mesh quality when faced with severe deformations. Copyright © 2023 John Wiley & Sons, Ltd.

KEY WORDS: high-order; curved mesh; mesh optimization; edge flip; edge collapse; deforming domains

## 1. INTRODUCTION

High-order methods have received considerable interest in the computational science community in the last two decades. This is due to their potential to deliver highly accurate solutions at a comparable cost to traditional low-order methods. In fact, for certain classes of problems — such as turbulent flow — it is widely believed that high-order methods will be *required* for accurate and grid-converged solutions [1]. Much of the great promise of high-order methods is due their ability to handle complex geometries with unstructured meshes. These methods require high-order (curved) meshes to ensure the curved geometry is represented

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with sufficient accuracy to enable the high-order convergence rate of these methods. However, the existence (or rather, lack thereof) of robust and automated high-order mesh generation procedures is still one of the main bottlenecks in the widespread adoption of these methods in the design process. To this end, the high-order community has identified high-order mesh generation as a topic deserving of considerable research attention and as one of the top pacing items in high-order method research [2, 3]. One might say curved meshing first distinguished itself as a distinct research topic within mesh generation in 2015, the first year it had its own dedicated session at the International Meshing Roundtable [4]. In terms of available software, Gmsh is a widely used free software with the capability to generate and display curved meshes for elements of arbitrarily high-order [5]. Up until fairly recently, there were no commercial high-order mesh generators available before Pointwise released support for this in 2019 [6]. We refer the reader to the introduction of [7] for a recent survey of the current state of affairs of high-order meshing research and additional references.

Most straight sided mesh generation procedures proceed in two steps: 1) the initial creation of a mesh from geometry through a method like advancing front, Delaunay or octree, and 2) a mesh optimization post-processing step to improve element quality. There is a large body of literature on these mesh quality improvement procedures. While those works may differ in their notions of optimality and specific techniques used, they largely agree mesh optimization via node movement and discrete, localized element operations are the best way to perform mesh quality improvement. Freitag and Ollivier-Gooch [8] consider various node smoothing procedures and face and edge swapping operations, and show that each mechanism fails to significantly improve the mesh quality when used individually but result in very high quality meshes when combined. They offer a variety of empirical recommendations on how to do so and demonstrate the effectiveness of their schedules on a variety of tetrahedral mesh geometries. De Cougny and Shephard [9] combine edge collapses and splitting operations for the purpose of mesh adaptation. Klinger and Shewchuk [10] consider a richer set of operations in an effort to take these ideas to an extreme to find the highest quality mesh, assuming that speed is not the highest priority. We refer the reader to Section 3 of their paper where they survey many of the commonly (as well as lesser used) discrete mesh operations. These are some of the "classic" papers in the field for mesh quality improvement for linear meshes, and while the focus of this paper is on curved meshes, the field of linear mesh quality improvement is still highly active. Some more recent contributions include, but are certainly not limited to directions like new discrete operations to reach better local minima [11, 12], and the inclusion of parallelization [13, 14, 15].

Now the question is how to extend these ideas to curved meshes. Existing work [16, 17, 18, 19] on high-order node smoothing primarily takes the approach of optimization-based minimization of some energy functional or objective function that quantifies high-order mesh distortion. The key issues here are the need for mesh distortion metrics for high-order elements [20, 21] and a procedure for high-order untangling [19, 22, 23]. In principle, some of the approaches to curved mesh generation such as the solution of a linear/nonlinear elasticity analogy [24, 25, 26, 27] or a PDE-based approach [28] could also be adapted for the purpose of high-order mesh smoothing. There is practically no literature on the use of local element operations for curved meshes. To the best of the authors' knowledge, the only existing work on high-order mesh operations [29] considers face and edge swaps for tetrahedral meshes with second-order  $(P^2)$  elements.

It is clear that further developments in these areas will be required to advance the current

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state of curved mesh generation and adaptivity. The difficulty of directly acting on high-order meshes is one possible explanation for why nearly all curved mesh generation approaches are of the *a priori* variety rather than direct methods [30] which proceed in the same two phases as most straight-sided mesh generators do. In the context of finite element solutions, solution quality is frequently dictated by the mesh element of the worst quality. Local mesh operations for curved meshes could be an inexpensive and direct way to deal with this and other applications such as the repair of deforming high-order domains occurring in time-dependent fluid simulations.



Figure 1: Straight sided mesh operations.

The main goal of this paper is to answer the question, "How we can extend local element operations for straight-sided meshes (Figure 1) to high-order triangular and tetrahedral meshes?" One key difference from the low-order case is that there are many valid ways to perform these operations due to the additional degrees of freedom afforded by the high-order nodes (Figure 2). So there is now an *optimal* way to perform these local operations by placing these additional nodes in a way to minimize a high-order Jacobian-based mesh distortion measure. In particular, this paper will discuss high-order face and edge swaps, edge collapses and edge splitting for triangular and tetrahedral meshes. The remainder of this article is organized as follows. In Section 2, we introduce the high-order Jacobian based distortion measure and its untangling capabilities. In Section 3, we describe the process of performing these operations, and discuss some practical details related to their implementation. In Section 4, we present multiple examples illustrating how these operations can be combined with highorder node smoothing to repair curved meshes that undergo severe deformations. Finally, in Section 5, we offer concluding remarks and directions for future research.

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Figure 2: Only one possible flip for straight-sided elements *(left)*, but many possible flips for high-order elements *(right)*.

## 2. DISTORTION MEASURE AND UNTANGLING

We describe the construction of the high-order distortion measure used in this work for node smoothing from a standard linear distortion measure, which is then regularized to allow for simultaneous smoothing and untangling [31, 32]. We also offer some comments on how this measure is used in our setting for optimization.

## 2.1. High-order distortion measure

We introduce the distortion measure for linear elements (2). This measure – known as the inverse mean-ratio shape measure – was first introduced in [33, 34]. This measure is widely used in the meshing community as a shape distortion metric due to its many desirable properties such as the standard invariance to translation, rotation, and scaling, but is also special as a geometric shape measure that also enjoys many of the desirable properties of an *algebraic* mesh quality metric [35]. In particular, this measure is shown to be convex [36] and well-suited for optimization.

As an aside, we note the metric we are about to introduce is geometric in nature, as are most metrics in use today, as opposed to being PDE or solution based. In the course of using mesh generation and adaptation in the numerical solution of PDEs, we point out that using a metric that is geometric in nature may not perfectly align with the ultimate goal of solving the PDE since the metric employed was not based on the PDE solution. This is addressed in [37] where the author discusses the notion of "bridging the gap between a priori quality metrics and solution-dependent metrics".

Consider a linear element  $E^I$  having the desired shape and size (the ideal element) and a corresponding element in physical space  $E^P$ . From these we can define a unique affine mapping  $\phi_E : E^I \to E^P$ . It is frequently convenient to introduce the notion of the master element  $E^M$  and define two additional mappings  $\phi_I, \phi_M$  (Figure 3a) that map from the master element to the ideal and physical element, respectively. We can define  $\phi_E$  in terms of the these mappings



Figure 3: Mappings between the master, ideal and physical elements: (a) linear case (left) and (b) high-order case (right).

as follows

$$\phi_E : E^I \xrightarrow{\phi_I^{-1}} E^M \xrightarrow{\phi_P} E^P. \tag{1}$$

The element shape distortion metric  $\eta$  is defined as

$$\eta(D\phi_E) = \frac{||D\phi_E||_F^2}{d|\sigma|^{2/d}}$$
(2)

where d is the spatial dimension,  $||\cdot||_F$  is the Frobenius norm, and  $\sigma = \det(D\phi_E)$ . Note that for linear elements, since  $\phi_E$  is affine, its Jacobian  $D\phi_E$  is constant. This distortion measure quantifies the deviation of the shape of the physical element  $E^P$  with respect to the ideal element  $E^I$ . This distortion measure is 1 for the ideal element and tends to infinity as the physical element degenerates. It is often convenient to define a corresponding quality measure q as

$$q = \frac{1}{\eta} \in [0, 1]. \tag{3}$$

For high-order elements, we can no longer apply (2) directly because the mapping  $\phi_E$  is no longer affine, so  $D\phi_E$  is no longer constant. Roca et al. describes an extension of the linear distortion measure to high order triangular [20] and tetrahedral [21] elements. The idea is that since the linear measure quantifies the local deviation between the ideal and physical elements, we can obtain a high-order element-wise distortion  $\hat{\eta}$  measure by integrating the linear measure on the physical high order element

$$\hat{\eta}(D\phi_E) = \left(\frac{1}{|t|} \int_t \eta^r (D\phi_E) \ dx\right)^{1/r} \tag{4}$$

where |t| is the area of the physical element and r = 2 in this work. Explicit expressions for the isoparametric mappings for the high-order case (Figure 3b) in terms of shape functions are given in [20].

## 2.2. Regularization and untangling

The distortion measure  $\eta$  needs to be modified to eliminate the singularity so an optimization procedure can recover from an invalid initial configuration. We use the regularization procedure

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introduced independently by [31, 32] around the same time, but follow the notation [32] which replaces  $\sigma$  in the denominator of (2) by

$$\sigma_{\delta}(\sigma) = \frac{1}{2} \left( \sigma + \sqrt{\sigma^2 + 4\delta^2} \right) \tag{5}$$

where  $\delta$  is a positive element-wise parameter. This regularized Jacobian  $\sigma_{\delta}(\sigma)$  is a monotonically increasing function of  $\sigma$  such that  $\sigma_{\delta}(0) = \delta$ , which tends to 0 when  $\sigma$  tends to  $-\infty$ , which allows us to overcome the vertical asymptote at  $\sigma = 0$  in the original measure. This parameter  $\delta$  is only set for a non-zero value if there exists an invalid element in the mesh under consideration, otherwise it is set to zero for a mesh with all valid elements. Using the regularized Jacobian, we can now modify (2) to obtain a linear distortion measure capable of *simultaneous* smoothing and untangling

$$\eta_{\delta}(D\phi_E) = \frac{||D\phi_E||_F^2}{d|\sigma_{\delta}|^{2/d}} \tag{6}$$

and use this to define a high-order regularized measure  $\hat{\eta}_{\delta}$  analogous to (4). Figures 5 and 6 of [19] provide a useful illustration of how this regularized distortion metric compares to the original for a simple example.

What remains is the issue of how to pick this positive elementwise regularization parameter  $\delta$ . We certainly require the original  $(\hat{\eta})$  and regularized  $(\hat{\eta}_{\delta})$  distortion measures to have nearby minima, so  $\delta$  needs to be sufficiently small. On the other hand,  $\delta$  has to be large enough to ensure that  $4\delta^2$  is significant compared to  $\sigma^2$  in the expression for  $\sigma_{\delta}$ . When this idea was originally introduced in [32], this parameter was chosen in a fairly ad hoc manner by testing a few values for a given initial tangled mesh. In [19], a heuristic was developed to choose a constant value of  $\delta$  for each element. Their setting deals with high-order mesh generation by curving the boundaries of a well-shaped straight-sided mesh and minimizing the regularized distortion metric. For them, in the mapping  $\phi_E$ , each physical element  $E^P$  in the curved mesh has the ideal element  $E^I$  as the corresponding element in the original straight-sided mesh. They propose to choose this parameter  $\delta$  solely based off information from the straight-sided ideal element. Defining  $\sigma^* = - \det \phi_I$  and imposing

$$\sigma_{\delta}(\sigma^*) = \frac{1}{2} \left( \sigma^* + \sqrt{(\sigma^*)^2 + 4\delta^2} \right) = \tau > 0 \tag{7}$$

for some given tolerance  $\tau$  implies

$$\delta(\sigma^*) = \frac{1}{2}\sqrt{(2\tau + |\sigma^*|)^2 - (\sigma^*)^2} = \sqrt{\tau^2 + \tau}|\sigma^*|.$$
(8)

They argue that  $\tau$  should be small compared to  $\sigma$  and select  $\tau = \alpha |\sigma^*|$ , giving the final value for  $\delta$  as

$$\delta(\sigma^*) = |\sigma^*| \sqrt{\alpha^2 + \alpha}.$$

The value  $\alpha = 10^{-3}$  is observed to work well in practice and accomplish the tradeoff required on the value of  $\delta$  (Figure 4). Our setting deals with high-order mesh smoothing, so we choose the ideal element  $E^I$  to always be the equilateral triangle/tetrahedron. For the unregularized distortion measure, the actual size of  $E^I$  is irrelevant since the measure is invariant to scaling. This is no longer true for the regularized distortion measure. So in order to maintain the assumption from their setting that each  $E^I$  and  $E^P$  are roughly of the same size, we need to scale the ideal element to have the same volume as each corresponding physical element and choose  $\delta$  accordingly.

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Figure 4: Representation of  $\sigma_{\delta}(\sigma)$  (reproduced from [19]).

### 2.3. Optimization

We need to aggregate the distortion measures for individual elements into a single quantity for a group of elements for optimization purposes. What we really want is to optimize for the worst element, since the distortion of the worst element in a mesh is often what drives the solution quality in the context of numerical simulation. But since minimizing the maximum distortion is difficult optimization-wise, we consider the normalized sum of squares of the elementwise distortion of each element  $e_i$  in a group of elements M

$$\hat{\eta}_{agg}(M) = \frac{1}{|M|} \sum_{e_i \in M} \hat{\eta}_{\delta} (D\phi_{e_i})^2 \tag{9}$$

where  $\hat{\eta}_{\delta}$  is the regularized high-order distortion measure obtained by combining (4) and (6)

$$\hat{\eta}_{\delta}(D\phi_E) = \left(\frac{1}{|t|} \int_t \eta^r_{\delta}(D\phi_E) \ dx\right)^{1/r}.$$
(10)

For smoothing over the entire mesh, we largely follow the localized approach given in Appendix A of [19], which considers a "Gauss-Seidel" like method which iteratively updates each node by a Newton step with backtracking line search until either the objective function or the amount of node movement satisfies some stopping criteria. Instead of looping over nodes, we loop over patches (Figure 5), which seems to help mitigate some of the "back and forth" the optimization procedure can empirically observed to become stuck in, which is also observed in [18] for a different high-order mesh smoothing procedure. We note this solver offers no guarantees for especially tangled or pathological meshes due to the issues with the regularization method which many authors have pointed out fails quite often in practice such as if the boundary of the mesh is also tangled [38], or the possible failure of the high-order Gauss quadrature rule to detect inversion at the corners of our high-order elements.

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Figure 5: Patch-based high-order mesh smoothing.

# 3. ELEMENT OPERATIONS

We describe how to perform the local element operations of edge/face swaps, edge collapses, and edge splitting for curved, simplex elements in 2D and 3D. The operations roughly proceed by freezing the boundary of the patch of affected elements, performing the operation as a "straight-sided one", and then simultaneously untangling and smoothing the result. We note some practical issues along with differences in the high-order setting.

## 3.1. Edge/face swaps

Once a group of elements is identified to perform edge/face swaps on, we first perform the straight-sided initial guess. This is done by freezing the boundary, discarding all the interior nodes and replacing them in the flipped configuration by the straight-sided information. This initial flip is extremely prone to tangling even when starting out with fairly well-shaped elements, which is more severe for the 3D case. By the high-order distortion measure (4), many high-order elements which seem valid by visual inspection with all interior nodes contained within the boundary may very well be invalid. Finally, we apply the simultaneous untangling and smoothing procedure described in the previous section to arrive at the curved flip (Figures 6 and 7). We decide to accept the resulting flip if it results in a lower aggregate distortion  $\hat{\eta}_{agg}$  from the original configuration. The normalization by the number of elements in the group in the definition is to allow comparison of the original and flipped configuration, since in 3D there are many topological changes which in general may alter the number of elements.

#### 3.2. Edge Collapses

Once an edge is identified to collapse, we first do the straight-sided initial guess. This is done by identifying the patch of elements that contains either endpoint of the edge being collapsed, freezing its boundary, discarding all the interior nodes, and replacing them by the straight-sided edge collapse to the midpoint of the original curved edge. Then we apply the simultaneous untangling and smoothing procedure to arrive the final curved collapse (Figures 8 and 9). If the edge identified for collapse is on the boundary of the domain, information will inevitably be lost but can be mitigated in the high-order case by performing projection based interpolation on the patch boundary by standard techniques encountered in hp-FEM methods [39].

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Figure 6: Edge flips in 2D: original elements *(left)*, initial straight-sided flip *(middle)*, curved flip *(right)*.



Figure 7: Face swaps in 3D: original elements *(left)*, initial straight-sided swap *(middle)*, curved swap *(right)*.



Figure 8: Edge collapses in 2D: original elements *(left)*, initial straight-sided collapse *(middle)*, curved collapse *(right)*.



Figure 9: Edge collapses in 3D: original elements *(left)*, initial straight-sided collapse *(middle)*, curved collapse *(right)*.

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#### 3.3. Edge Splitting

Depending on the setting, the edges to be split could either be directly identified or implied by choosing elements. Either way, once an element is identified to be split and the straight sided guess is performed according to one of the templates in 2D (Figure 10) or 3D (Figure 11), its neighbors must also be split by a straight-sided guess with the corresponding template to prevent hanging nodes. Then, we apply simultaneous untangling and smoothing on this group of elements to arrive at the final curved split (Figures 12 and 13). There are many more possible subdivision templates in 3D depending on the number of marked edges (Figure 11 of [9]), and that for any of these templates to determine how to split the edge(s), the amount of "smoothing" required after the edge splits applied to the straight sided guess can differ quite greatly and alter the overall efficiency of the method, but this is outside of the scope of this work.

## 4. NUMERICAL EXPERIMENTS

We demonstrate how local element operations for curved meshes can be combined with highorder mesh smoothing to maintain element quality and sizing for high-order meshes subject to severe deformations in both 2D and 3D. All the meshes used for the numerical experiments are generated by Gmsh [5] and are of polynomial degree four.

All the results have been obtained on a MacBook Pro with one dual-core Intel Core i7 CPU (Apple Inc., Cupertino, California, US), with a clock frequency of 3.0 GHz, and total memory of 16 GBytes. As a proof of concept, this code has been fully developed in MATLAB without using any additional toolbox. The code is not optimized, not parallel, and not compiled and the results are provided as an illustration of this concept.

## 4.1. Rotation

We first demonstrate how node smoothing combined with edge flips can maintain element quality under rotation. The 2D mesh is a circle-in-square mesh of 116 elements, where the circle is centered at the origin with radius 0.5 inside the square  $[-1, 1]^2$  (Figure 14 *left*). The analogous 3D mesh is a sphere-in-cube mesh of 3395 elements, where the sphere is centered at the origin with radius 0.5 inside the cube  $[-1, 1]^3$  (Figure 15 *left*). At each step, we rotate the nodes on the circle/sphere, perform high-order node smoothing while keeping the nodes on the circle/sphere fixed, apply curved flips according to Algorithm 1, and smooth again. Algorithm 1 constructs a priority queue of elements with higher distortion than a floating threshold value and considers all topologically possible flips for each element, accepting the one (if any) that results in the greatest improvement. It performs sweeps over all the elements with a reset floating threshold for each sweep until no further improvement is possible.

Rotation cannot exceed much more than a quarter turn with only node smoothing (Figures 14 and 15 *middle*). A straight sided mesh cannot rotate this far with smoothing alone; the presence of high order nodes grant us the flexibility to maintain valid (albeit highly stretched) elements for more severe deformations. It is well known in the straight-sided case that such greedy flip algorithms like the one used here converge to an optimal triangulation (in the sense of Delaunay) in the 2D case, but offers no such guarantees in higher dimensions [40].

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Figure 10: Element subdivision templates in 2D:  $1 \rightarrow 2$  (*left*) and  $1 \rightarrow 4$  (*right*).



Figure 11: Element subdivision templates in 3D:  $1 \rightarrow 2$  (*left*),  $1 \rightarrow 4$  (*middle*), and  $1 \rightarrow 8$  (*right*).



Figure 12: Edge splitting in 2D: original elements *(left)*, initial straight-sided split *(middle)*, curved split *(right)*.



Figure 13: Edge splitting in 3D: original elements *(left)*, initial straight-sided split *(middle)*, curved split *(right)*.

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2D rotation can continue indefinitely, and we can reasonably attain a good aggregate quality under rotation for the 3D case, but some poorly shaped elements emerge as evidenced by the growing maximum element distortion (Table I); this issue is no doubt magnified by our use of very coarse meshes. We also report the aggregate mesh quality (9), and the numerical results also illustrate a fundamental issue of mesh smoothing discussed at the beginning of Section 2.3. It turns out even if the aggregate distortion of the mesh is very low, this can obscure the fact that there are elements of very low quality present as well. This can also be remedied with more sophisticated mesh cleanup procedures featuring more specialized element operations such as compound operations [41] and sliver exudation [42], but such issues are not the focus of this work. Also note that here we do not nodes on the boundary or the circle/sphere. Allowing these nodes to slide on the fixed boundaries while smoothing the entire mesh would certainly be an improvement to the procedure.

## 4.2. Coarsening

Though this example does not deal with mesh deformation, we demonstrate how curved collapses can be combined with high-order node smoothing to coarsen a mesh (Figure 16). We begin with a rather fine 2D mesh with 690 elements of the square  $[-1,1] \times [-1,1]$  with a circular hole at the origin with radius 0.5 (Figure 16, top left). We apply multiple passes of curved edge collapses to coarsen the mesh until convergence according to a chosen uniform size function (Algorithm 2). In this case we choose the ideal edge length  $L_0 = 0.1$  as a parameter far enough from the original edge length just to illustrate the procedure to induce collapses. We also coarsen elements on the interior boundary with projection by interpolation in the  $L^2$  error minimizing sense to maintain an accurate representation of the circular boundary with fewer elements.

#### 4.3. Translation

We demonstrate the combination of all three curved element operations with mesh smoothing to maintain both element quality and uniform sizing. The 2D mesh is a circle-in-rectangle mesh of 314 elements, where the circle is centered at (-1, 0) and has radius 0.5 inside the rectangle  $[-2,2] \times [-1,1]$ . The analogous 3D mesh is a sphere-in-prism mesh of 3395 elements, where the sphere is centered at (-1, 0, 0) with radius 0.5 inside the prism  $[-2, 2] \times [-1, 1] \times [-1, 1]$ . At each step, we translate the nodes on the circle/sphere, perform high-order node smoothing while keeping the nodes on the circle/sphere fixed, apply all three types of local mesh operations according to Algorithm 4, and smooth again. Algorithm 4 applies sweeps of splits (Algorithm 3) and collapses (Algorithm 2) until convergence, and then applies sweeps of flips (Algorithm 1) until convergence. In terms of the parameters for both Algorithms 2 and 3, the ideal edge length is set to be  $L_0 = \operatorname{avg}(L)$ , where L is the list of computed original edge lengths. This choice is made simply because after the translation, we want the final mesh to resemble our original mesh, which was roughly uniform and with the same element sizes. As an aside, one could imagine introducing anisotropy to the mesh through these local element operations through a specified size function which would specify the ideal lengths  $L_0$  at any point in space, but this is beyond the scope of the work.

Translation cannot make it more than halfway across the prism with only node smoothing (Figures 17 and 18). With the addition of flips, we can make it across the entire domain, but this results in elements of highly variable size. This can be remedied with the addition of



Figure 14: 2D rotation with flips: original mesh (left), quarter turn with smoothing only (middle), quarter turn with smoothing and flips (right).



Figure 15: 3D rotation with flips: original mesh (left), quarter turn with smoothing only (*middle*), quarter turn with smoothing and flips (*right*).

	2D		31	)
	max	agg	max	agg
Original mesh	1.167	1.141	2.499	1.669
Smoothing only	7.876	5.327	15.531	8.490
With flips	1.197	1.179	4.086	1.826

Table I: Mesh D	istortion of Fi	igures 14 and 15
	2D	3D
		100 G ##

collapses and splitting to maintain a uniform size, which allows us to practically maintain the original element sizes and qualities after translation across the entire domain (Table II).

# 5. CONCLUSION AND FUTURE WORK

We extend the local element operations of edge flips, edge collapses and edge splitting to the case of curved meshes. Our numerical examples demonstrate how these local operations can be combined with node smoothing to successfully perform mesh adaptation directly on curved

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Figure 16: 2D coarsening with collapses: original mesh (*top left*), one sweep of collapses (*top right*), five sweeps of collapses (*bottom left*), final mesh (*bottom right*).

meshes to maintain high element quality and desired sizing. Curved meshes can naturally deal with more severe deformations than straight sided ones on much coarser meshes due to the higher number of degrees of freedom, and mesh adaptation can be applied to take advantage of this fact to deal with more severe deformations. These operations are more expensive to perform in the curved setting; for straight-sided meshes they amount to a simple permutation of the existing data and require no node smoothing. However, in the big picture this is a small price to pay, since in any practical setting these local operations deal with a relatively small number of elements when used as an alternative to a far more costly procedure such as remeshing.

We see two avenues of future work to fully unlock the utility of these operations towards mesh adaptation for curved meshes. More efficient mesh optimization procedures combining these ingredients need to be developed; in this work, we use fairly naive and greedy algorithms which are straightforward, but far from optimal. As noted in [5], mesh optimization procedures

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Figure 17: 2D translation: original mesh (top left), smoothing only (top right), smoothing with flips (bottom left), smoothing with flips, collapses, and splitting (bottom right).

currently are based on "black magic: even if the ingredients required to construct a mesh optimization procedure are well known ... there is no known best recipe, i.e., no known optimal way of combining these." In the 15 years since this statement was made, there have certainly been many improvements in the "ingredients", but there doesn't seem to have much work focused solely on the "recipes". The large body of existing work on these mesh optimization procedures is mainly concerned with achieving the best possible mesh quality without much concern for speed. This is a significant limitation in the curved setting, since node smoothing is the main bottleneck in these procedures and is much more expensive in the high-order case. As Klinger and Shewchuk note in [10], mesh *optimization* procedure, and "if the barrier of speed can be overcome, the need to write separate programs for mesh generation and mesh improvement might someday disappear." This is particularly relevant for the issue of high-order mesh generation, since the limited number of procedures currently in use are nearly all of the *a priori* variety rather than direct methods. Surely, this is in no small part due to our limited ability to currently perform mesh adaptation directly on curved meshes.

The real challenge in the high-order setting is the ability to perform smoothing quickly and untangling robustly. It is worth emphasizing just how much of the cost comes from the smoothing procedure. In this work we prescribe the mesh motion, which must be followed by a smoothing of the entire mesh, a pass of local mesh operations, and and then another smoothing of the entire mesh. The two contributions to the cost of each local mesh operation is a simple permutation of indices in the mesh data structure for the straight sided guess followed by smoothing. Since the cost of the former is practically negligible, virtually all of the cost comes from the mesh smoothing procedure, made even more expensive in a *high-order* mesh. While

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Figure 18: 3D translation: original mesh (top left), smoothing only (top right), smoothing with flips (bottom left), smoothing with flips, collapses, and splitting (bottom right).

Table II: Mesh Distortion of Figures $17$ and $18$				
	2D		3D	
	max	$\operatorname{agg}$	$\max$	$\operatorname{agg}$
Original mesh	1.1821	1.0473	2.2466	1.7481
Smoothing only	2.8378	2.3771	7.3829	3.8409
With flips	2.0643	1.2075	3.8427	2.1742
Flips collapses and splitting	1 1891	1.0498	3 0003	1 9666

Flips, collapses, and splitting 1.1821 1.0498 3.0003 1.9666 the approach for untangling considered in this work of the regularization of a mesh distortion metric has the advantage that it is relatively simple, it is very dependent on a heuristically chosen parameter and many authors have pointed out that it fails quite often in practice for various reasons, such as if the boundary of the mesh is also tangled or if the inversion primarily occurs at the corners of the elements [19, 38]. Another limitation of these optimization-based approaches is that they are largely based on the special choice or modification of some objective function and lack specialized solvers. A promising idea is the extension of some of the ideas in [28, 43, 44] towards a fully unstructured, high-order Winslow-based smoothing and untangling procedure. Winslow-based methods have been shown to be very robust in practice and do not require the choice of any empirical parameters. Each iteration of these methods essentially

From here, we can revisit the vast amount of literature on mesh optimization procedures almost exclusively for the straight-sided case to better understand the differences and limitations in the curved setting, which we know very little about. Further developments in

requires an elliptic solve, for which there exists highly robust and efficient solvers (i.e. multigrid)

and would make a huge difference in efficiency for the high-order setting.

this area would also remove one of the constraints on the use of high-order elements for the so called arbitrary Lagrangian-Eulerian (ALE) simulation framework for moving meshes with large deformations [45].

Data Availability Statement: Research data are not shared.

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# APPENDIX

Algorithm 1: Application of Curved Flips

if dim = 2D then tmin = 1.5else if dim = 3D then tmin = 2Input curved mesh M with elements  $\{e_i\}$ Set distortion threshold  $T = \min(tmin, \min(D)), D(i) = \text{distortion of } e_i$ Create priority queue  $P = \{(e_j, D(j)), \forall j \ D(j) > T \text{ descending}\}$ while true do for i in P do if dim = 2D then consider all possible  $2 \rightarrow 2$  flips with  $e_i$ else if dim = 3D then consider all possible  $2 \rightarrow 3, 3 \rightarrow 2, 4 \rightarrow 4$  flips with  $e_i$ if best of topologically possible flips improves  $\hat{\eta}_{agg}$  then Update the mesh  ${\cal M}$ Remove all involved elements from the queue P and renumber end end if no flips done in sweep then return M end Recompute  $\{e_i\}, T$  and P  $\quad \text{end} \quad$ 

 Algorithm 2: Application of Collapses

 Set ideal edge length as parameter  $L_0$  

 Input curved mesh M with edges  $\{E_j\}$  

 Compute list of edge lengths L 

 Create priority queue  $P = \{(E_k, L(E_k)), \forall k \mid \frac{L_0}{L(E_k)} > 1.5 \text{ descending}\}$  

 while true do

 for i in P do

 Perform curved collapse on edge  $e_i$  and update the mesh M 

 Remove all edges in collapsed patch from P and renumber

 end

 if no collapses done in sweep then

  $\mid$  return M 

 end

 Recompute  $\{E_j\}, L$  and P 

 end

 Recompute  $\{E_j\}, L$  and P 

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$\mathbf{A}$	lgorithm	3:	App	lication	of	Spl	litting
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Set ideal edge length as parameter  $L_0$ Input curved mesh M with edges  $\{E_i\}$ Compute list of edge lengths LCreate priority queue  $P = \{(E_k, L(E_k)), \forall k \ \frac{L_0}{L(E_k)} > 1.5 \ \text{descending}\}$ while true do for i in P do Identify elements K containing edge  $E_i$ for j in K do Identify all edges of element  $e_i$  in queue P to choose template Perform curved split on element  $e_i$  and update the mesh MRemove all edges in collapsed patch from P and renumber end end if no splits done in sweep then return Mend Recompute  $\{E_i\}$ , L and P end

Algorithm 4: Application of all operations: flips, coarsening, and splitting

Input curved mesh M with elements  $\{e_i\}$  and edges  $\{E_j\}$ Perform sweeps of splits (Algorithm 3) Perform sweeps of collapses (Algorithm 2) Perform smoothing of the entire mesh MPerform sweeps of flips (Algorithm 1) Perform smoothing of the entire mesh M**return** M

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